

# Optimal Design of Motion Control Systems



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## What is Optimal Design

A. System components must have the capability to move in minimum time

**Hardware capability + trajectory planning**

B. System must use its capabilities and be tuned for optimum performance

When the motion time, T, gets shorter:

- acceleration increases by  $1/T^2$
- current and power dissipation increase
- motor overheats

**Motion time should be long enough to avoid damages to the motors**

Another formulation of objective:

Perform a move within time T, while minimizing the motor temperature

## Design Steps

- Best motor selection (**Higher utility, yet no overheating**)
- Best coupling between motor and load
- Best velocity profile

## Optimality

- Rotate a motor by an angle  $\theta$  and stop within T
- Select the best velocity profile that results in minimum motor temperature  
**minimum power dissipation**

## Optimal Velocity Profile

**From Optimal Control Theory**

The optimal velocity profile is a parabola

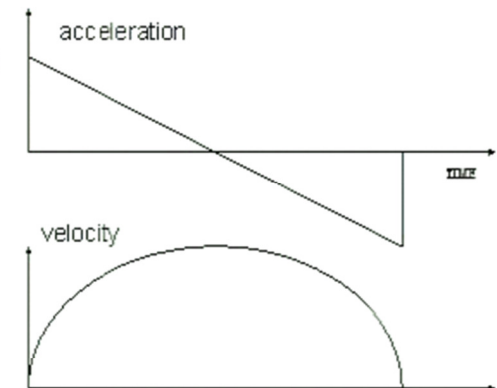
**Minimum energy dissipation**

Energy dissipation per step is

$$E = \frac{12rJ^2\theta^2}{K_t^2 T^3}$$

Where:

- r = armature resistance
- J = moment of inertia
- $\theta$  = rotation angle
- $K_t$  = torque constant
- T = motion time



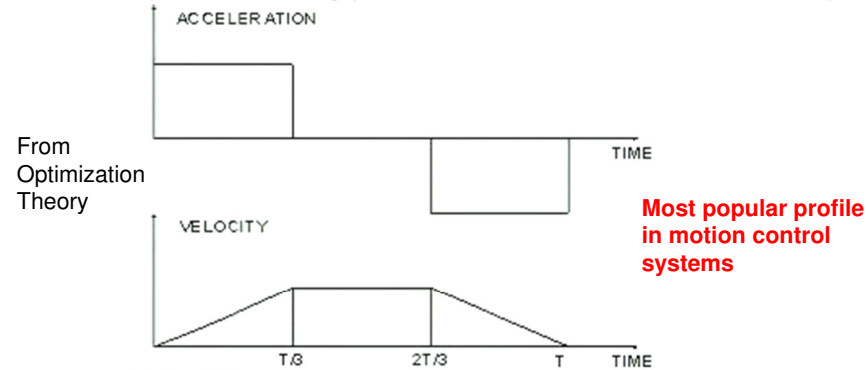
**High currents at the ends saturates the Amp. Thus, parabolic profiles are not commonly used**

**Optimum profile is not practical**

## Suboptimal Profile: Trapezoidal

- The closest to the optimal (parabolic) is the trapezoidal velocity profile. Then, what's the optimum combination of acceleration - uniform velocity - deceleration ?

The best trapezoidal velocity profile is where the three intervals are equal

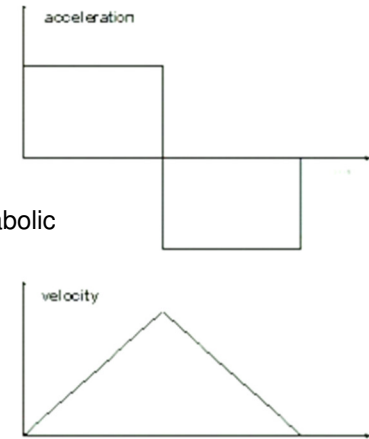


Here  $E = \frac{13.5rJ\theta^2}{K_t^2 T^3}$   $\implies$  12% increase in power dissipation compared to parabolic velocity profile <sup>5</sup>

## Suboptimal Profile: Triangular

- Acceleration Bang-Bang
  - Not optimal in terms of power dissipation

Triangular profile



33.3% increase in power dissipation compared to parabolic velocity profile

$$E = \frac{16rJ^2\theta^2}{K_t^2 T^3}$$

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## Optimal Coupling

Examples of coupling:

- Gear box
- Reduction by pulleys
- Belt and pulley
- Rack and pinion
- Lead screw

**Coupling is optimal when motor and load inertia are matched**



**minimum temperature rise in the motor**

Objective: Select coupling ratio to minimize motor temperature

Solution: Select coupling ratio to achieve inertial match between motor and reflected load

Rotary Load  $\implies$

Motor inertia - $J_m$
Load inertia - $J_{L0}$
Reduction ratio - $N$
Reflected Load - $J_L = J_{L0} / N^2$
Inertial match - $J_m = J_{L0} / N^2$
Optimal Coupling - $N = \sqrt{J_{L0} / J_m}$

## Optimal Coupling

Ex: A motor with inertia  $J_m = 0.0002 \text{ kgm}^2$  drives 0.5kg load through a pulley. Calculate the pulley radius for optimal performance

Motor inertia -  $J_m$

Load mass -  $M$

Pulley radius -  $R$

Reflected Load -  $J_L = MR^2$

Inertial match -  $J_m = MR^2$

$$R = \sqrt{\frac{J_m}{M}} = \sqrt{\frac{0.0002}{0.5}} = 0.002m$$

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## Non-optimal Coupling Torque and Power Penalty

What if we cannot achieve optimal coupling ?

If optimal coupling is  $N$ , and

actual coupling ratio is  $n=D*N$ , where

$D$ = Deviation from optimum

then,

Torque penalty:  $RT = \frac{1}{2} (D+1/D)$  → Unnecessary torque developed

Power dissipation penalty:  $PT = \frac{1}{4} (D+1/D)^2$

← Unnecessary power dissipation

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## Non-optimal Coupling

Ex: A motor with  $J_m=0.0002\text{kgm}^2$  direct drives a load  $J_L=0.005\text{kgm}^2$ . Check whether direct drive is optimal, if not, calculate the torque penalty and power penalty

Optimal reduction is  $N=5$  ←  $J_m = J_L \times \frac{1}{N^2}$   
 If direct drive is used  $n=1$ , then  $D=0.2$  ←  $D = \frac{n}{N}$   
 Torque penalty:  $RT=2.6$   
 Power penalty:  $RP=6.76$

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## Optimum Motor Selection Quality Factor

Objective: Select the motor that results in minimum temperature

Answer: Select a motor with minimum:

$$Q = \frac{r J_m R_{th}}{K_t^2} \quad \Rightarrow \quad \text{Calculate } Q \text{ for all candidate motors. } Q \text{ is proportional to temperature rise}$$

Where:

$R$  – armature resistance

$J_m$  – motor inertia

$R_{th}$  – thermal resistance of motor

$K_t$  – torque constant

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## Optimum Motor Selection

Design procedure:

Motion requirements are defined

Motor is selected

Optimal coupling between motor and load is used.

Ex: Following three motors satisfy the design requirement of a certain application. Select the most optimal motor

Motor	A	B	C
$r (\Omega)$	4.0	2	1.8
$J_m [\text{Kg} \cdot \text{m}^2]$	0.00012	0.0002	0.0005
$R_{th} [^\circ\text{C} / \text{W}]$	3	2.2	1.8
$K_t [\text{Nm} / \text{A}]$	0.08	0.12	0.15

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# Flexible Control Systems

- Centralized Control Systems
- Distributed Control Systems
- Flexible-Distributed Control Systems

# Centralized Control Systems

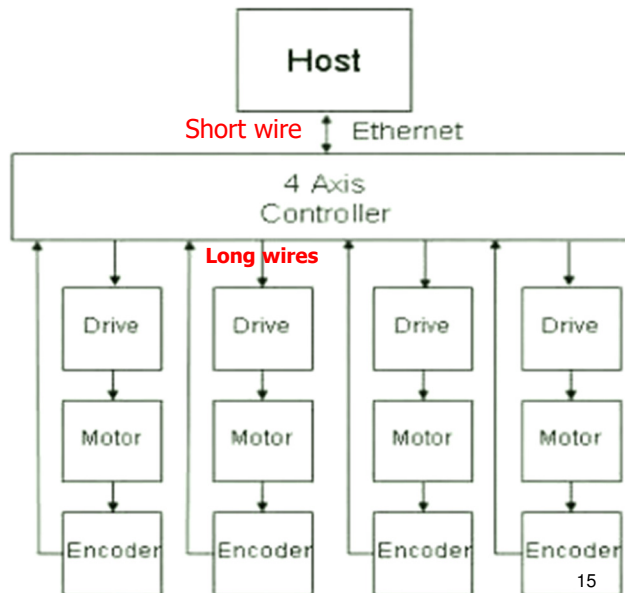
- All motors are controlled directly by one multi-axis motion controller
  - E.g. **DMC-2183: 8axis, Ethernet**
- Advantages
  - Simple communication – host communicates with a single motion controller
  - Easy programming – information about all motors is present in one controller
  - Coordinated motion is performed by motion controller
  - Low controller cost per axis
- Disadvantages
  - Long wiring – wires from drives, encoders and switches must be connected to the central controller

# Centralized Control Systems cntd..

Sometimes, machine design dictates that controllers be distributed through the machine to minimize wiring. In this case, central control is not an option

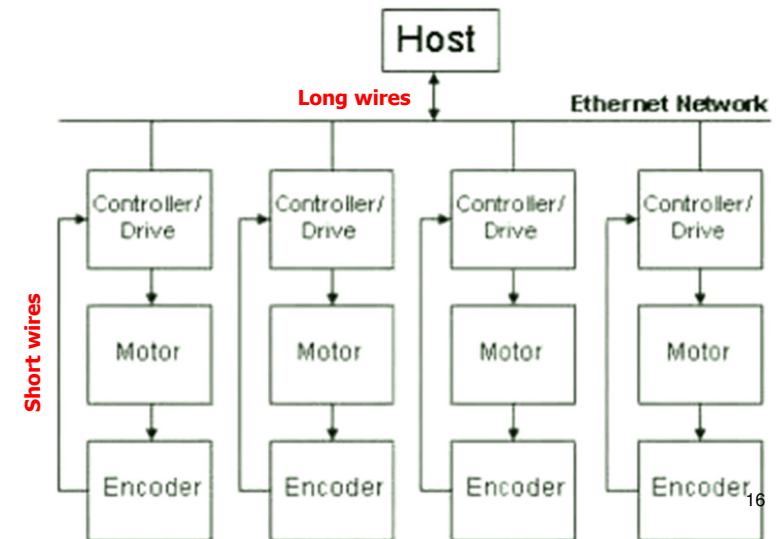


**Distributed Control Systems**



# Distributed Control Systems

Motors are controlled by several motion controllers that are distributed throughout the machine



# Distributed Control Systems

Typical distributed control systems use single-axis controllers or drives

**Advantages:**

- Short wires – eliminates associated problems

**Disadvantages:**

- Complicated communication
- Complex motion programming
- High cost of network components
- High cost of controller per axis

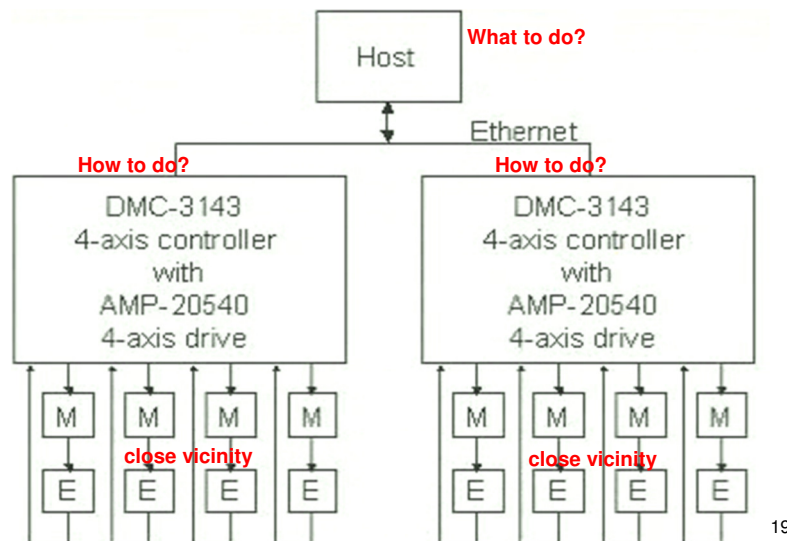
# Ethernet Based Distributed Control

Popular and widely used in Industry. A cost effective choice

Ethernet offers advantages over other networks

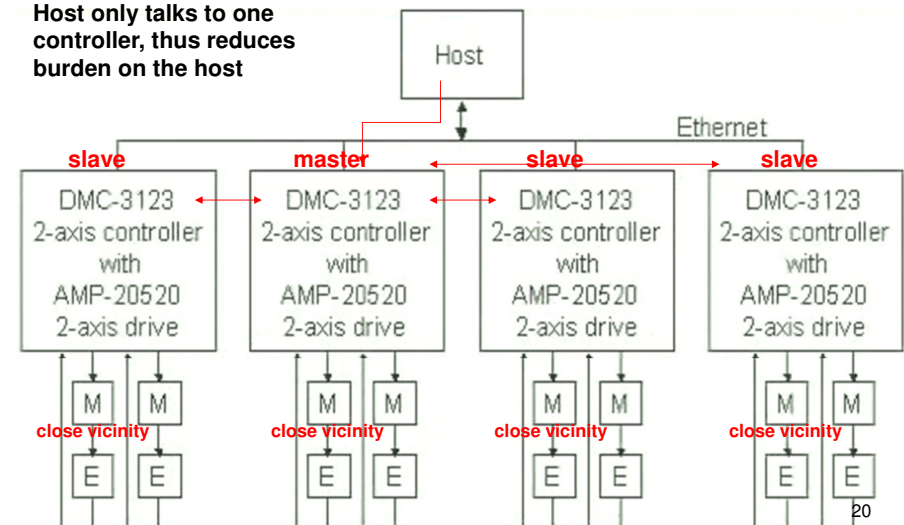
1. Lower cost of network components
2. TCP/IP protocol is well known
3. Network allows mixing components such as controller, I/O devices, camera, etc.
4. Ethernet systems are non-deterministic, and data sent more frequently than 1 msec may be delayed. Since communication is at a high-level between the host and the controller (commands sent less frequently than 1 msec) the non-deterministic nature of Ethernet is not an issue.

# Flexible Distributed Control Systems: Centralized Islands



# Flexible Distributed Control Systems: Centralized Islands

Host only talks to one controller, thus reduces burden on the host



# Flexible Distributed Control Systems

- Combines the advantages of central and distributed control systems
- Compared to distributed control systems, FDCS is

Flexible

Reduces host communication burden

Performs coordinated motion

Lower cost

} multiple axis controllers

- Flexible-distributed control is not restricted to single-axis motion controllers.
- The user divides his system into islands with each island having from 1 to 8 axes
- This gives designer *flexibility*.
- Example- An 8-axis machine can use:

**4# of 2-axis controllers, or**

**2# of 4-axis controllers, or**

**2# of 3-axis controller+1# of 2-axis controller**